Tutorial—Five Quick Problems

Tutorials will use the **Standard User Mode** to highlight the simplicity in using this software. Advanced Users may want to use the **Power User Mode** and examples are shown at the end of each Tutorial.

All Tutorials are shown with the US unit system.

Problem 2

Let's extend Problem 1 by adding beam support and load information. Specifically, assume that the beam will be steel, 6 ft long, fixed at both ends, with a maximum concentrated load of 2000 lbf at the center. You want to determine the deflection and stress values.

Open the Roark's Formulas for Excel Explorer and select **Table A.1**, **Case 3** (hollow rectangular cross-section). Then select **Table 8.1**, **Case 1d** and add this to the previous case selection.



Click the Standard User Mode button to continue.

Once loaded into Excel the Worksheet looks as follows:

| | A | В | C | D | E | Free Free | G | H |
|----|-----------------|------------------|--------------------|---|-----------|----------------|------------------|---|
| 1 | Roark's Form | mulas for Ex | cel | | | | | |
| 2 | Appendix: Pro | perties of a Pla | ne Area TABLE | A.1 | | | | |
| 3 | Table A.1 - Pro | perties of Secti | ons Pages 802 | - 812 | | | | |
| 4 | Case: 3 Hollow | Rectangle | | | | | | |
| 5 | Chapter Eight: | Beams; Flexu | re of Straight Ba | ars TABLES 8.1 - 8.13 | | | | |
| 6 | 1. Concentrate | d Intermediate l | _oad | | | | | |
| 7 | Case: 1d Fixed | I, Fixed | | | | | | |
| 8 | _ | | | | | 0 | | 0 |
| 9 | Но | llowrectangle | | Left end fixed, right end fixed | Concentra | ted intermedia | ate load | |
| 10 | | 2 y ₂ | | | Y | 14 | M | |
| 11 | 1 | 1 | | N | | a ——— ' | ^м в 🔪 | |
| 12 | - | d. | | - N | _ | | | |
| 14 | d 1- | H + | 1 | - ў · · · · · · · · · · · · · · · · · · | | - | | 0 |
| 15 | | | y | | | € _A | R ₈ | |
| 16 | | | i — | В — м | A | | | |
| 17 | _ | 2 | • | | <u>t</u> | 1 | | |
| 18 | 4 | b | | | RA | | 1.1 | |
| 19 | | | | 0 | | 0 | | 0 |
| 20 | | | | | | | | |
| 21 | Input | Value | Unit | Comment | | | | |
| 22 | axis | 1 | | Neutral Axis (1,2) | | | | |
| 23 | t1b | 1 | in | Side b | | | | |
| 24 | t1bi | 0.75 | in | Hollow Side bi | | | | |
| 25 | t1d | 2 | in | Side d | | | | |
| 26 | t1di | 1.5 | in | Hollow Side di | | | | |
| 27 | matnum | 17 | | Material number (see material table) | | | | |
| 28 | L | 72 | in | Length of beam | | 1 | | |
| 29 | a | 36 1000 | in Ing | Load distance from left end | | | | |
| 30 | VV V | 1000 | ioi in | Load Sample dictance, x, from left and | | | | |
| 32 | ^ | JU | 101 | | | | | |
| 32 | Output | Value | Linit | Comment | | | | |
| 34 | Δ | 0.875 | inA7 | Area A | | | | |
| 35 | t1v | 1 | in | Centroid to Extremity | | | | |
| 36 | 1 | 0.455729167 | in^4 | Area moment of inertia | | | | |
| 37 | 1%c | 0.455729167 | in^3 | Elastic section modulus | | | | |
| 38 | t1r | 0.721687836 | in | Radius of Gyration, r | | | | |
| 39 | Z | 0.578125 | in^3 | Plastic Section Modulus, Z | | | | |
| 40 | SF | 1.268571429 | | Shape Factor, SF | | | | |
| 41 | t | 2 | in | Depth | | | | |
| 42 | ĸ | 0.26796875 | in^4 | Torsional Stiffness Constant, K | | | | |
| 43 | Q 01 | 0.765625 | in/3 | Shear Stress Constant, Q | | | | |
| 44 | moti | U.3828125 | µп^5 М. 87 617" | Silver Stress Constant, Long Side, UT | | | | |
| 45 | F | 29000000 | nei | Voung's Modulus | | | | |
| 40 | | 23000000 | lhf | Transverse shear at v | | | | |
| 48 | M | 9000 | lhf-in | Bending moment at x | | | | |
| 49 | theta | 0 | rad | Slope Angle at x | | | | |
| 50 | Y | -0.147092808 | in | Deflection at x | | | | |
| 51 | sty | 19748.57143 | psi | Max fiber stress at extremity at location x | | | | |
| 52 | RA | 500 | lbf | Vertical reaction at left end | | | | |
| 53 | MA | -9000 | lbf-in | Bending moment at left end | | | | |
| 54 | thetaA | 0 | rad | Slope Angle at left end | | | | |
| 55 | уА | 0 | in | Deflection at left end | | | | |
| 56 | RB | 500 | lbf | Vertical reaction at right end | | | | |
| 57 | MB | -9000 | lbf-in | Bending moment at right end | | | | |
| 58 | thetaB | 0 | rad | Slope angle at right end | | | | |
| 59 | ув | 0 | In | Deflection at right end | | | | |
| 60 | | | | | | | | |



Now, enter the cross section dimensions in the Worksheet. Simply move to the Value column of the relevant variable and enter the data.

| 1 |
|----------------|
| 1.5 in |
| 1.3125 in |
| 2.75 in |
| 2 in |
| |

Note the revised Excel Worksheet.





5

Power User Mode Approach

The following series of problems uses the Power User Mode approach in the Roark's Formulas for Excel Explorer.

Problem 2

Let's extend Problem 1 by adding beam support and load information. Specifically, assume that the beam will be steel, 6 ft long, fixed at both ends, with a maximum concentrated load of 2000 lbf at the center. You want to determine the deflection and stress values.

Open the Roark's Formulas for Excel Explorer and select **Table A.1**, **Case 3** (hollow rectangular crosssection). Then select **Table 8.1**, **Case 1d** and add this to the previous case selection. Click the **Power User Mode** button.



This combines the cross section variables and formulas with the beam loading formulas and opens an Input form having the combined Variables, Tables and Plots tabs as shown below.

First, enter the cross section dimensions as done in Problem 1.

| 1 |
|-----------------------|
| 1.5 in |
| 1.3125 |
| 2.75 in |
| 2 in |
| 6 ft (1.828 m) |
| |

Enter a length of 6 m and then change the unit from ft to m simply highlight the unit and select the appropriate unit.

Roark's Formulas for Excel supports unit conversions for a wide range of quantities used in the application.

| | | | | | Variables |
|-----|--------|--------|--------|--------|--------------------------------------|
| Sta | Input | Name | Output | Units | Comments |
| | | | | | Left end fixed, right end fixed |
| | | | | | Table 8.1: Case 1 - Roark's Formulas |
| | | | | | Concentrated Intermediate Load |
| | | matnum | | | Material number (see material table) |
| | | matl | | | Material description |
| | 1.8288 | L | | m 🔻 | Length of beam |
| | | а | | in | Load distance from left end |
| | | W | | lbf | Load |
| | | E | | psi | Young's Modulus |
| | | I | | in^4 | Area moment of inertia, I |
| | | z | | in | Neutral axis to stress point, z |
| | | х | | in | Sample distance, x, from left end |
| | | V | | lbf | Transverse shear at x |
| | | M | | lbf-in | Bending moment at x |
| | | theta | | rad | Slope Angle at x |

Enter the inputs for the remaining variables:

| Load distance from left end | 36 in |
|-----------------------------|-----------------|
| Load | 2000 lbf |
| Young's Modulus | 3E7 psi |

Distance x is the location along the beam at which the shear, moment, slope, deflection, and stress will be calculated and reported. By default, the model automatically solves for the two ends. By entering a value for x, you also get a solution at the desired location.

If you don't remember the Young's Modulus for steel, you can check the Material Properties Table, which is automatically loaded with each case. You can look at the Material Properties Table by clicking the Tables Tab and selecting Materials.

The table appears as shown below.

| 3 Hollow Rectangle <table a.1<cross-sectio<="" th=""><th>n Properti</th><th>ies<roark's excel;1d="" fixed<="" for="" formulas="" th=""><th>, Fixed<case< th=""><th>1<table th="" x<=""></table></th></case<></th></roark's></th></table> | n Properti | ies <roark's excel;1d="" fixed<="" for="" formulas="" th=""><th>, Fixed<case< th=""><th>1<table th="" x<=""></table></th></case<></th></roark's> | , Fixed <case< th=""><th>1<table th="" x<=""></table></th></case<> | 1 <table th="" x<=""></table> |
|--|------------|--|--|-------------------------------|
| Variables Tables Plots | | | | 1 |
| Materials | | Materials | | |
| | matn | matl | G | E 🔺 |
| | | | Shear | Young's |
| | | | Modulus | Modulus |
| | | METALS | | |
| | 1 | Aluminum - cast, pure | 3700000 | 9000000 |
| | 2 | Aluminum - cast, 220 T4 | 3550000 | 9500000 🦳 |
| | 3 | Aluminum - wrought, 2014 T6 | 4000000 | 10600000 |
| | 4 | Aluminum - wrought, 6061 T6 | 3750000 | 1E7 |
| | 5 | Beryllium copper | 7000000 | 19000000 |
| | 6 | Brass - naval | 5500000 | 15000000 |
| | 7 | Bronze - Phosphor, A.S.T.M. B159 | 6500000 | 15000000 |
| | 8 | Cast iron, gray, no. 20 | 6700000 | 14000000 |
| | 9 | Cast iron, gray, no. 30 | 6700000 | 15200000 |
| | 10 | Cast iron, gray, no. 40 | 6700000 | 18300000 |
| | 11 | Cast iron, gray, no. 60 | 6700000 | 1900000 |
| | 12 | Cast iron - malleable | 8800000 | 2600000(🚽 |
| | • | i- · ·· | | · |
| Variables Tables Plots | | Solve | tup Layout | Cancel |

The material number appears in the left column. This is the value you input for the variable matnum. The program will then retrieve the required values from the row of the table. We could use material number 17 for steel in this example.

Click the Solve button and the results are shown.

| | | | | | Variables |
|-----|--------|--------|--------|--------|--------------------------------------|
| Sta | Input | Name | Output | Units | Comments |
| | | | | | Left end fixed, right end fixed |
| | | | | | Table 8.1: Case 1 - Roark's Formulas |
| | | | | | Concentrated Intermediate Load |
| | | matnum | | | Material number (see material table) |
| | | matl | | | Material description |
| | 1.8288 | L | | m | Length of beam |
| | 36 | а | | in | Load distance from left end |
| | 2000 | W | | lbf | Load |
| | 3E7 | E | | psi | Young's Modulus |
| | | I | 1.8979 | in^4 | Area moment of inertia, I |
| | | z | ' | in | Neutral axis to stress point, z |
| | 36 | х | | in | Sample distance, x, from left end |
| | | V | 1000 | lbf | Transverse shear at x |
| | | M | 18000 | lbf-in | Bending moment at x |
| | | theta | 0 | rad | Slope Angle at x |

Visit www.uts.com and www.roarksformulas.com

Let's change the beam material from steel to aluminum. Select the Tables tab and view the Materials Table again. Notice that materials 1-4 are various types of aluminum. Use number 4. Select the Variables tab and enter 4 as an input for the variable named matnum.

Blank the current value for the Young's Modulus E.

Solve and view the results.

| | | | | | Variables |
|-----|--------|--------|---------|--------|--------------------------------------|
| Sta | Input | Name | Output | Units | Comments |
| | | | | | Left end fixed, right end fixed |
| | | | | | Table 8.1: Case 1 - Roark's Formulas |
| | | | | | Concentrated Intermediate Load |
| | 4 | matnum | | | Material number (see material table) |
| | | matl | "Alumin | | Material description |
| | 1.8288 | L | | m | Length of beam |
| | 36 | а | | in | Load distance from left end |
| | 2000 | W | | lbf | Load |
| | | E | 1E7 | psi | Young's Modulus |
| | | I | 1.8979 | in^4 | Area moment of inertia, I |
| | | z | ' | in | Neutral axis to stress point, z |
| | 36 | x | | in | Sample distance, x, from left end |
| | | V | 1000 | lbf | Transverse shear at x |
| | | M | 18000 | lbf-in | Bending moment at x |
| | | theta | 0 | rad | Slope Angle at x |

The change in materials has significantly increased the deflection at x.

Now select Setup Layout and load the model in the Excel Worksheet.



